

# Experimental Analysis on Fabricated Parabolic Solar Collector with Various Flowing Fluids and Pipe Materials

Ankur Geete\*, Somesh Kothari\*\*‡, Rahul Sahu\*\*\*, Prateek Likhar\*\*\*\*, Ankit Saini\*\*\*\*\*, Anil Singh\*\*\*\*\*

\*Mechanical Engineering Department, Sushila Devi Bansal College of Technology, Indore, Madhya Pradesh, India

\*\* Mechanical Engineering Department, Sushila Devi Bansal College of Technology, Indore, Madhya Pradesh, India

\*\*\* Mechanical Engineering Department, Sushila Devi Bansal College of Technology, Indore, Madhya Pradesh, India

\*\*\*\* Mechanical Engineering Department, Sushila Devi Bansal College of Technology, Indore, Madhya Pradesh, India

\*\*\*\*\* Mechanical Engineering Department, Sushila Devi Bansal College of Technology, Indore, Madhya Pradesh, India

\*\*\*\*\* Mechanical Engineering Department, Sushila Devi Bansal College of Technology, Indore, Madhya Pradesh, India

(ankur\_geete@yahoo.co.in, someshkothari27@gmail.com, rahulsahu8878@gmail.com, prateeklikharr@gmail.com, ankitsaini.as@gmail.com, anilsingh0994@gmail.com)

‡ Corresponding Author; Ankur Geete, Sushila Devi Bansal College of Technology, Indore, Madhya Pradesh, India, Tel: +91 9165438182, ankur\_geete@yahoo.co.in

*Received: 01.07.2016 Accepted: 16.08.2016*

**Abstract-** This work is accomplished on fabricated parabolic solar collector. Numbers of experiments are performed on collector at different operating conditions with tracking mode III. Operating conditions are; copper-water arrangement, copper-engine oil arrangement, mild steel-water arrangement and mild steel-engine oil arrangement at 12:30PM, at 01:30PM, 02:30PM and at 03:30PM. Then fluid outlet temperature, inside/outside pipe surface temperatures, heat transfer rate, instantaneous efficiency, convective heat transfer coefficient, absorbed flux and collector efficiency factor are experimentally found. Inlet exergy is also analysed for this solar collector with and without considering Sun's cone angle. Finally this work can be concluded as – If fluid temperature difference decreases then heat gain rate increases by increasing mass flow rate of fluid. When aperture width, pipe length and mass flow rate of fluid increase then inlet exergy increases. Maximum instantaneous efficiency (84.28%) is achieved at 12:30PM with copper-water combination at 0.0166kg/sec flow rate. Maximum and minimum temperature difference (59°C and 3°C) of fluid are found at 12:30PM and at 02:30PM respectively. Maximum inlet exergy (2839.854W) is achieved with 1.75m pipe length and 3.0m aperture width but minimum inlet exergy (858.392W) is found with 0.75 m aperture width and 2.0 m pipe length.

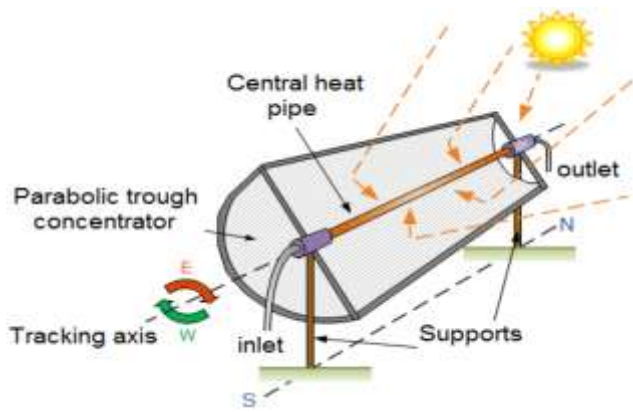
**Keywords** Parabolic solar collector, solar energy, instantaneous efficiency, collector efficiency factor.

## 1. Introduction

This research work is related with solar energy. Many research works have been done by researchers in this field because solar energy is the most significant non-

conventional sources of energy and it is non-polluting therefore it helps to decreasing the greenhouse effect. This work is done on parabolic solar collector. Parabolic solar collectors are thermal collectors that are curved as parabola in two dimensions but straight in one dimension. The curved

surfaces have polished mirrors which reflect solar energy or thermal energy. This solar energy is focused at focus axis of the parabola and then thermal energy is utilized for different purposes; like power plants, chemical industries etc. A simplest layout parabolic solar collector is as shown in figure 1. From layout, mirrors which are assembled with curved surface reflect sunlight on the pipe. Pipe is mounted at focus axis of the parabola and it contains fluid which is heated to high temperature by the use of thermal energy. Many researchers have done lot of work in the field of solar energy. They have optimized solar energy systems with different methods or different ways [2-8]. In this research work parabolic solar collector is fabricated and experiments are performed. Fabricated collector is shown in figure 2. From various tracking modes, mode III tracking mode is chosen in this research work. This mode has focal axis in North-South direction and its orientation is horizontal [1].



**Figure 1** – Schematic layout of cylindrical parabolic solar collector



**Figure 2** – Fabricated cylindrical parabolic solar collector with glass plate

In this experimental work, 64 number of experiments are performed with different input parameters or conditions. Here water and engine oil are used as flowing fluids while mild steel and copper are selected as pipe materials. With experiments fluid outlet temperatures, heat transfer rates, instantaneous efficiencies, absorbed fluxes, collector efficient factors and inlet exergies are found. Whole experimental work is done on typical summer months of April and May at Sushila Devi Bansal College of Technology, Indore, Madhya Pradesh, India (Latitude - 22°43'4.51"N and Longitude - 75°49'59.88"E). 12:30PM,

01:30PM, 02:30PM and 03:30PM are selected time intervals on which all observations are taken.

**2. Methodology**

Experimental analyses are performed on parabolic solar collector with different fluids and with different pipe materials. All observations are taken at fix time intervals and that time instantaneous beam radiations for per unit surface area are; at 12.30 PM - 526.90 W/m<sup>2</sup>, at 01.30 PM – 529.0 W/m<sup>2</sup>, at 02.30 PM - 537.20 W/m<sup>2</sup> and at 03.30 PM - 472.80 W/m<sup>2</sup> [1]. Following mathematical equations are used for research work. Instantaneous efficiency [1, 3, 5, 7],

$$\eta_{ib} = (Q_u) / (I_b R_b W L) \tag{1}$$

Here  $Q_u$  is rate of heat transfer in Watts,  $I_b R_b$  is instantaneous beam radiation on per unit area of the surface in W/m<sup>2</sup>,  $W$  is aperture width in meters and  $L$  is length of the pipe in meters. Useful heat gain rate [1, 3, 4, 7],

$$Q_u = m_f C_p (T_{fo} - T_{fi}) \tag{2}$$

Here  $m_f$  is mass flow rate of fluid which is flowing in the pipe in kg/sec,  $C_p$  is specific heat of fluid in J/kgK,  $T_{fo}$  and  $T_{fi}$  are fluid temperatures at inlet and outlet respectively in Kelvin. Convective heat transfer [1],

$$Q_u = h_f A_s (T_p - T_{fo}) \tag{3}$$

Here  $h_f$  is convective heat transfer coefficient in W/m<sup>2</sup>K,  $A_s$  is surface area of pipe in m<sup>2</sup>,  $T_p$  and  $T_{fo}$  are temperatures of pipe and outlet fluid respectively in Kelvin.

Concentration ratio is the ratio of effective aperture area to pipe surface area. It is represented by  $C$ . Here  $W$  is aperture width in meters,  $D_o$  is pipe outer diameter in meter and  $L$  is effective length in meter, Concentration ratio [1, 3, 10],

$$C = \{(W - D_o) L\} / \{\pi D_o L\} \tag{4}$$

Absorbed flux [1],

$$S = \{I_b R_b \rho \gamma (\tau \alpha)_b + I_b R_b (\tau \alpha)_b [D_o / (W - D_o)]\} \tag{5}$$

Here  $\rho$  is reflectivity of the collector surface, some amount of reflected radiation is intercepted by the pipe which is known as intercept factor and it is denoted by  $\gamma$ ,  $\tau$  is transmissivity and  $\alpha$  is absorbtivity. Conductive heat transfer through the pipe [8],

$$Q_u = \{[2 \pi K L (T_c - T_p)] / [\ln (D_o/D_i)]\} \tag{6}$$

Here  $K$  is thermal conductivity of the pipe material in W/mK,  $T_c$  are  $T_p$  are pipe surface temperature at outlet and inlet respectively in Kelvin. Amount of heat transfer ( $Q_u$ ) can also be written with overall heat loss coefficient ( $U_l$ ), concentration ratio ( $C$ ) and absorbed flux ( $S$ ) as [1, 12],

$$Q_u = [S - (U_l / C) (T_p - T_a)] (W - D_o) L \tag{7}$$

Collector efficiency factor [1],

$$F' = [U_l \{(1 / U_l) + [D_o / (D_i h_f)]\}]^{-1} \tag{8}$$

Focus point for cylindrical parabolic collector is found by mathematical equation given below [1, 3].

$$F = (W^2) / (16 C) \tag{9}$$

Exergy inlet for solar collector is the summation of exergy available in solar radiation and exergy taken by flowing fluid. Exergy analysis for solar collector is calculated by following equations which is given by Petela [9]. Exergy inlet and maximum useful work available from solar radiation are calculated by equations 10, 11 and 12 where equation 11 gives maximum useful work without considering Sun's cone angle but equation 12 give maximum useful work with consideration of Sun's cone angle which is equal to 0.005 radian [11].

$$\Psi_{inlet} = m_f C_p (T_{fi} - T_{fo}) - m_f C_p (\ln T_{fi} - \ln T_{fo}) + I_b R_b (W L_c) \Psi \quad (10)$$

$$\Psi = 1 - (4 T_0 / 3 T_s) + (1/3) (T_0 / T_s)^4 \quad (11)$$

$$\Psi' = 1 - (4 T_0 / 3 T_s) (1 - \cos \bar{\theta})^{0.25} + (1/3) (T_0 / T_s)^4 \quad (12)$$

Here  $\Psi_{inlet}$  is exergy at inlet in Watts and  $\Psi$  is maximum useful work,  $T_0$  and  $T_s$  are ambient temperature and Sun surface temperature (which is near about 5800K) in Kelvin.

**Results –**

For experimental work, cylindrical parabolic solar collector is fabricated. Important dimensions are; (a) aperture width (W) of the collector is 1.25 m, (b) length of collector and absorber pipe is 2.5 m, (c) inner diameter ( $D_i$ ) of hollow pipe is 0.028 m, (d) outer diameter ( $D_o$ ) of hollow pipe is 0.032 m and concentration ratio (C) is 12.12. Assumed

values for calculations are - specular reflectivity of glass which is mounted on concentration surface is 0.39, intercept factor for absorber pipe is 0.95, transmissivity and absorptivity are 0.32 and 0.29 respectively. All the experimental results are found with four different combinations; combinations are - copper-water, copper-engine oil, mild steel-engine oil and mild steel-water. Fluid and surface temperatures are measured by digital thermometer. Results are shown in tabulated form in tables 1-16 with different time intervals.

Inlet exergy analyses are also done by considering Sun's cone angle and without considering Sun's cone angle at various operating conditions. After that, inlet exergy results are shown in figures 3, 4, 5 and 6. Inlet exergies at different aperture widths with 0.012kg/sec mass flow rate and 32°C inlet temperature for copper-oil arrangement are shown in figure 3. Inlet exergies at different pipe lengths with 0.0019kg/sec mass flow rate and 32°C inlet temperature for mild steel-water arrangement are shown in figure 4. Inlet exergies at different aperture widths and pipe lengths with 0.0049kg/sec mass flow rate for mild steel-oil arrangement are shown in figure 5. Inlet exergies at different mass flow rates of flowing fluid with 31°C inlet temperature for copper-water arrangement are shown in figure 6.

**Table 1 – At 12:30PM with Copper Water combination ( $I_b R_b$  526.90 W/m<sup>2</sup>)**

Sr. No.	Inlet Parameter					Outlet Parameter				
	$m_f$ (kg/sec)	$T_{fi}$ (°C)	$T_c$ (°C)	$T_{fo}$ (°C)	$T_p$ (°C)	$Q_u$ (W)	$D_{ib}$ (%)	$h_f$ (W/m <sup>2</sup> K)	S (W/m <sup>2</sup> )	F'
1	0.00326	31	91	64	90.99	449.68	27.31	75.801	19.401	0.69
2	0.01163	31	100	51	99.98	972.26	59.04	90.313	19.401	0.57
3	0.0166	31	95	51	94.97	1387.76	84.28	143.592	19.401	0.58
4	0.01247	31	100	49	99.98	938.24	56.98	83.731	19.401	0.56

**Table 2 – At 01:30PM with Copper Water combination ( $I_b R_b$  529.0 W/m<sup>2</sup>)**

Sr. No.	Inlet Parameter					Outlet Parameter				
	$m_f$ (kg/sec)	$T_{fi}$ (°C)	$T_c$ (°C)	$T_{fo}$ (°C)	$T_p$ (°C)	$Q_u$ (W)	$D_{ib}$ (%)	$h_f$ (W/m <sup>2</sup> K)	S (W/m <sup>2</sup> )	F'
1	0.01079	31	94	52	93.98	947.14	57.29	102.647	19.478	0.59
2	0.00415	31	95	52	94.98	364.28	22.04	38.550	19.478	0.61
3	0.0165	31	95	45	94.98	965.58	58.41	87.897	19.478	0.55
4	0.00781	31	98	49	97.98	587.62	35.55	54.574	19.478	0.58

**Table 3 – At 02:30PM with Copper Water combination ( $I_b R_b$  537.2 W/m<sup>2</sup>)**

Inlet Parameter					Outlet Parameter				
-----------------	--	--	--	--	------------------	--	--	--	--

Sr. No.	$m_f$ (kg/sec)	$T_{fi}$ (°C)	$T_c$ (°C)	$T_{fo}$ (°C)	$T_p$ (°C)	$Q_u$ (W)	$D_{ib}$ (%)	$h_f$ (W/m <sup>2</sup> K)	$S$ (W/m <sup>2</sup> )	$F'$
1	<b>0.00279</b>	<b>31</b>	<b>88</b>	55	87.99	279.89	16.67	38.595	19.78	0.66
2	<b>0.01505</b>	<b>31</b>	<b>88</b>	38.5	87.99	471.82	28.11	43.37	19.78	0.54
3	<b>0.01589</b>	<b>31</b>	<b>80</b>	37.4	79.99	425.08	25.32	45.408	19.78	0.53
4	<b>0.00967</b>	<b>31</b>	<b>52</b>	40.3	51.98	497.17	29.62	260.322	19.78	0.65

**Table 4** – At 03:30PM with Copper Water combination ( $I_bR_b$  472.8 W/m<sup>2</sup>)

Inlet Parameter					Outlet Parameter					
Sr. No.	$m_f$ (kg/sec)	$T_{fi}$ (°C)	$T_c$ (°C)	$T_{fo}$ (°C)	$T_p$ (°C)	$Q_u$ (W)	$D_{ib}$ (%)	$h_f$ (W/m <sup>2</sup> K)	$S$ (W/m <sup>2</sup> )	$F'$
1	<b>0.00615</b>	<b>31</b>	<b>57</b>	40.6	56.99	246.78	16.67	68.483	17.409	0.60
2	<b>0.0099</b>	<b>31</b>	<b>57</b>	36.2	56.99	215.186	14.56	47.079	17.409	0.55
3	<b>0.00331</b>	<b>31</b>	<b>38.6</b>	36.4	38.59	74.713	5.06	154.647	17.409	0.48
4	<b>0.00563</b>	<b>31</b>	<b>37.8</b>	35.1	37.79	96.487	6.53	162.704	17.409	0.14

**Table 5** – At 12:30PM with Copper Engine Oil combination ( $I_bR_b$  526.90 W/m<sup>2</sup>)

Inlet Parameter					Outlet Parameter					
Sr. No.	$m_f$ (kg/sec)	$T_{fi}$ (°C)	$T_c$ (°C)	$T_{fo}$ (°C)	$T_p$ (°C)	$Q_u$ (W)	$D_{ib}$ (%)	$h_f$ (W/m <sup>2</sup> K)	$S$ (W/m <sup>2</sup> )	$F'$
1	<b>0.0063</b>	<b>32</b>	<b>132</b>	67	131.99	350.374	21.28	24.527	19.401	0.63
2	<b>0.0032</b>	<b>32</b>	<b>132</b>	58	131.99	132.205	8.03	8.128	19.401	0.70
3	<b>0.012</b>	<b>32</b>	<b>132</b>	70	131.98	724.584	44.01	53.183	19.401	0.62
4	<b>0.0072</b>	<b>32</b>	<b>132</b>	59	131.99	308.902	18.76	19.254	19.401	0.61

**Table 6** – At 01:30PM with Copper Engine Oil combination ( $I_bR_b$  529.0 W/m<sup>2</sup>)

Inlet Parameter					Outlet Parameter					
Sr. No.	$m_f$ (kg/sec)	$T_{fi}$ (°C)	$T_c$ (°C)	$T_{fo}$ (°C)	$T_p$ (°C)	$Q_u$ (W)	$D_{ib}$ (%)	$h_f$ (W/m <sup>2</sup> K)	$S$ (W/m <sup>2</sup> )	$F'$
1	<b>0.0058</b>	<b>32</b>	<b>129</b>	79	128.98	433.161	26.20	39.421	19.478	0.68
2	<b>0.0013</b>	<b>38</b>	<b>129</b>	74	128.99	74.365	4.5	6.152	19.478	0.89
3	<b>0.0102</b>	<b>38</b>	<b>127</b>	64.2	126.99	424.644	25.69	30.768	19.478	0.62
4	<b>0.0025</b>	<b>38</b>	<b>127</b>	64	126.99	103.285	6.25	7.459	19.478	0.77

**Table 7** – At 02:30PM with Copper Engine Oil combination ( $I_bR_b$  537.2 W/m<sup>2</sup>)

Inlet Parameter					Outlet Parameter					
Sr. No.	$m_f$ (kg/sec)	$T_{fi}$ (°C)	$T_c$ (°C)	$T_{fo}$ (°C)	$T_p$ (°C)	$Q_u$ (W)	$D_{ib}$ (%)	$h_f$ (W/m <sup>2</sup> K)	$S$ (W/m <sup>2</sup> )	$F'$

1	<b>0.0034</b>	<b>32</b>	<b>62</b>	55	61.99	124.26	7.4	80.796	19.78	0.87
2	<b>0.00659</b>	<b>32</b>	<b>68</b>	65	67.98	345.56	20.58	525.278	19.78	0.92
3	<b>0.0066</b>	<b>32</b>	<b>65</b>	63	64.98	325.109	19.37	742.154	19.78	0.94
4	<b>0.025</b>	<b>32</b>	<b>55</b>	50	54.96	715.05	42.59	652.595	19.78	0.79

**Table 8** – At 03:30PM with Copper Engine Oil combination ( $I_b R_b$  472.8 W/m<sup>2</sup>)

Sr. No.	Inlet Parameter					Outlet Parameter				
	<b>m<sub>f</sub></b> (kg/sec)	<b>T<sub>fi</sub></b> (°C)	<b>T<sub>c</sub></b> (°C)	<b>T<sub>fo</sub></b> (°C)	<b>T<sub>p</sub></b> (°C)	<b>Q<sub>u</sub></b> (W)	<b>D<sub>ib</sub></b> (%)	<b>h<sub>f</sub></b> (W/m <sup>2</sup> K)	<b>S</b> (W/m <sup>2</sup> )	<b>F'</b>
1	<b>0.0043</b>	<b>32</b>	<b>55</b>	38	54.99	40.996	2.77	10.972	17.409	0.77
2	<b>0.00218</b>	<b>32</b>	<b>52</b>	40	51.99	27.712	1.88	10.507	17.409	0.56
3	<b>0.0073</b>	<b>32</b>	<b>55</b>	35	54.99	34.799	2.36	7.916	17.409	0.62
4	<b>0.008</b>	<b>32</b>	<b>52</b>	42	51.99	127.12	8.6	57.852	17.409	0.71

**Table 9** – At 12:30PM with Mild Steel Water combination ( $I_b R_b$  526.90 W/m<sup>2</sup>)

Sr. No.	Inlet Parameter					Outlet Parameter				
	<b>m<sub>f</sub></b> (kg/sec)	<b>T<sub>fi</sub></b> (°C)	<b>T<sub>c</sub></b> (°C)	<b>T<sub>fo</sub></b> (°C)	<b>T<sub>p</sub></b> (°C)	<b>Q<sub>u</sub></b> (W)	<b>D<sub>ib</sub></b> (%)	<b>h<sub>f</sub></b> (W/m <sup>2</sup> K)	<b>S</b> (W/m <sup>2</sup> )	<b>F'</b>
1	<b>0.00544</b>	<b>32</b>	<b>48</b>	45	47.99	305.93	18.58	465.04	19.401	0.81
2	<b>0.0084</b>	<b>32</b>	<b>41</b>	38	40.99	210.672	12.79	319.917	19.401	0.58
3	<b>0.0382</b>	<b>32</b>	<b>40.5</b>	36	40.48	638.704	38.79	647.758	19.401	0.38
4	<b>0.0586</b>	<b>32</b>	<b>39.5</b>	36	39.47	979.792	59.51	1281.303	19.401	0.31

**Table 10** – At 01:30PM with Mild Steel Water combination ( $I_b R_b$  529.0 W/m<sup>2</sup>)

Sr. No.	Inlet Parameter					Outlet Parameter				
	<b>m<sub>f</sub></b> (kg/sec)	<b>T<sub>fi</sub></b> (°C)	<b>T<sub>c</sub></b> (°C)	<b>T<sub>fo</sub></b> (°C)	<b>T<sub>p</sub></b> (°C)	<b>Q<sub>u</sub></b> (W)	<b>D<sub>ib</sub></b> (%)	<b>h<sub>f</sub></b> (W/m <sup>2</sup> K)	<b>S</b> (W/m <sup>2</sup> )	<b>F'</b>
1	<b>0.016</b>	<b>32</b>	<b>60</b>	45	59.94	869.44	52.59	264.023	19.478	0.61
2	<b>0.0011</b>	<b>32</b>	<b>58</b>	43	57.99	50.578	3.06	15.342	19.478	0.89
3	<b>0.0023</b>	<b>32</b>	<b>43.4</b>	40.4	43.37	80.758	4.89	122.554	19.478	0.87
4	<b>0.0013</b>	<b>32</b>	<b>40</b>	38.4	39.98	34.778	2.1	98.953	19.478	0.64

**Table 11** – At 02:30PM with Mild Steel Water combination ( $I_b R_b$  537.2 W/m<sup>2</sup>)

Sr. No.	Inlet Parameter					Outlet Parameter				
	<b>m<sub>f</sub></b> (kg/sec)	<b>T<sub>fi</sub></b> (°C)	<b>T<sub>c</sub></b> (°C)	<b>T<sub>fo</sub></b> (°C)	<b>T<sub>p</sub></b> (°C)	<b>Q<sub>u</sub></b> (W)	<b>D<sub>ib</sub></b> (%)	<b>h<sub>f</sub></b> (W/m <sup>2</sup> K)	<b>S</b> (W/m <sup>2</sup> )	<b>F'</b>
1	<b>0.0019</b>	<b>32</b>	<b>107</b>	54	106.94	174.724	10.41	15.00	19.78	0.67
2	<b>0.0188</b>	<b>32</b>	<b>55</b>	44	54.69	943.008	56.17	390.738	19.78	0.62

3	<b>0.0105</b>	<b>32</b>	<b>50</b>	40	49.86	351.12	20.92	159.857	19.78	0.59
4	<b>0.00298</b>	<b>32</b>	<b>46</b>	37	45.97	62.282	3.71	31.488	19.78	0.96

**Table 12** – At 03:30PM with Mild Steel Water combination ( $I_bR_b$  472.8 W/m<sup>2</sup>)

Sr. No.	Inlet Parameter					Outlet Parameter				
	<b>m<sub>f</sub></b> (kg/sec)	<b>T<sub>fi</sub></b> (°C)	<b>T<sub>c</sub></b> (°C)	<b>T<sub>fo</sub></b> (°C)	<b>T<sub>p</sub></b> (°C)	<b>Q<sub>u</sub></b> (W)	<b>D<sub>ib</sub></b> (%)	<b>h<sub>f</sub></b> (W/m <sup>2</sup> K)	<b>S</b> (W/m <sup>2</sup> )	<b>F'</b>
1	<b>0.00214</b>	<b>32</b>	<b>48</b>	40	47.97	71.562	4.84	40.707	17.409	0.83
2	<b>0.0087</b>	<b>32</b>	<b>41</b>	36	40.95	145.464	9.85	132.44	17.409	0.49
3	<b>0.00828</b>	<b>32</b>	<b>44</b>	37	43.94	173.052	11.71	112.538	17.409	0.55
4	<b>0.00281</b>	<b>32</b>	<b>40.3</b>	37.5	40.27	64.602	4.37	105.006	17.409	0.82

**Table 13** – At 12:30PM with Mild Steel Engine Oil combination ( $I_bR_b$  526.90 W/m<sup>2</sup>)

Sr. No.	Inlet Parameter					Outlet Parameter				
	<b>m<sub>f</sub></b> (kg/sec)	<b>T<sub>fi</sub></b> (°C)	<b>T<sub>c</sub></b> (°C)	<b>T<sub>fo</sub></b> (°C)	<b>T<sub>p</sub></b> (°C)	<b>Q<sub>u</sub></b> (W)	<b>D<sub>ib</sub></b> (%)	<b>h<sub>f</sub></b> (W/m <sup>2</sup> K)	<b>S</b> (W/m <sup>2</sup> )	<b>F'</b>
1	<b>0.0024</b>	<b>33</b>	<b>134</b>	92	133.4	225.002	13.66	24.376	19.401	0.76
2	<b>0.0164</b>	<b>33</b>	<b>106</b>	76	105.97	1120.563	68.05	170.073	19.401	0.71
3	<b>0.0091</b>	<b>33</b>	<b>84</b>	55	83.9	318.118	19.32	49.919	19.401	0.66
4	<b>0.0086</b>	<b>33</b>	<b>74</b>	68	73.97	478.289	29.05	363.275	19.401	0.87

**Table 14** – At 01:30PM with Mild Steel Engine Oil combination ( $I_bR_b$  529.0 W/m<sup>2</sup>)

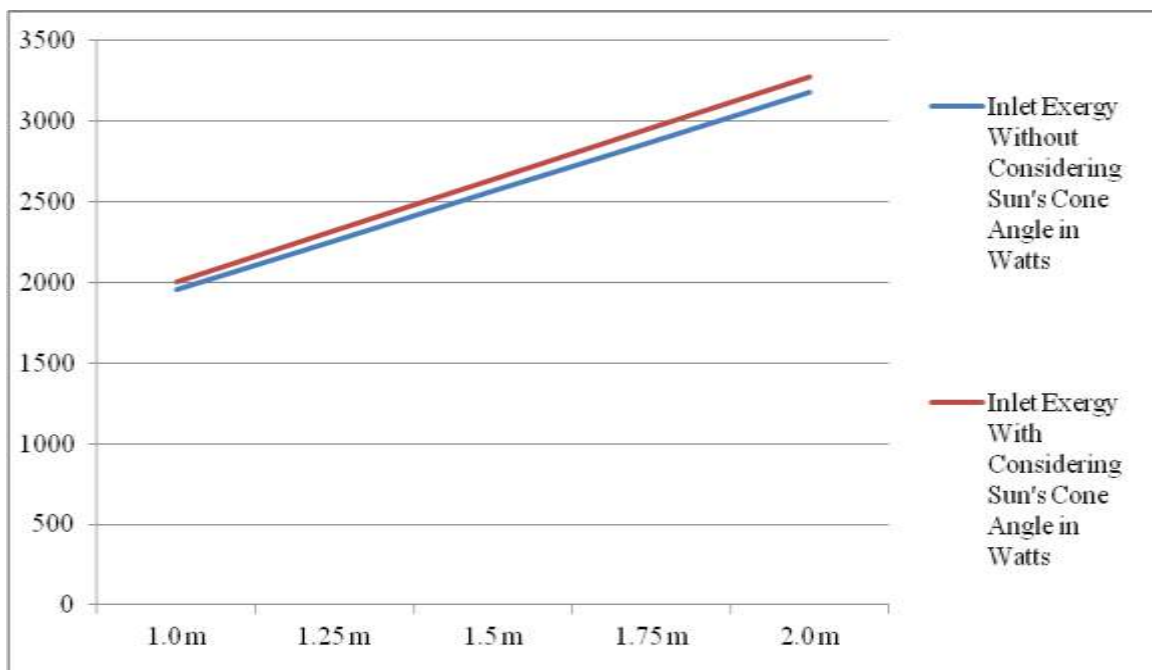
Sr. No.	Inlet Parameter					Outlet Parameter				
	<b>m<sub>f</sub></b> (kg/sec)	<b>T<sub>fi</sub></b> (°C)	<b>T<sub>c</sub></b> (°C)	<b>T<sub>fo</sub></b> (°C)	<b>T<sub>p</sub></b> (°C)	<b>Q<sub>u</sub></b> (W)	<b>D<sub>ib</sub></b> (%)	<b>h<sub>f</sub></b> (W/m <sup>2</sup> K)	<b>S</b> (W/m <sup>2</sup> )	<b>F'</b>
1	<b>0.0046</b>	<b>52</b>	<b>88</b>	60	87.973	58.475	3.54	9.502	19.478	0.99
2	<b>0.015</b>	<b>41</b>	<b>70</b>	63.5	69.8	536.288	32.44	376.004	19.478	0.85
3	<b>0.0064</b>	<b>42</b>	<b>68</b>	57	67.94	152.544	9.23	63.109	19.478	0.82
4	<b>0.013</b>	<b>32</b>	<b>66</b>	45	65.89	268.541	16.24	58.195	19.478	0.63

**Table 15** – At 02:30PM with Mild Steel Engine Oil combination ( $I_bR_b$  537.2 W/m<sup>2</sup>)

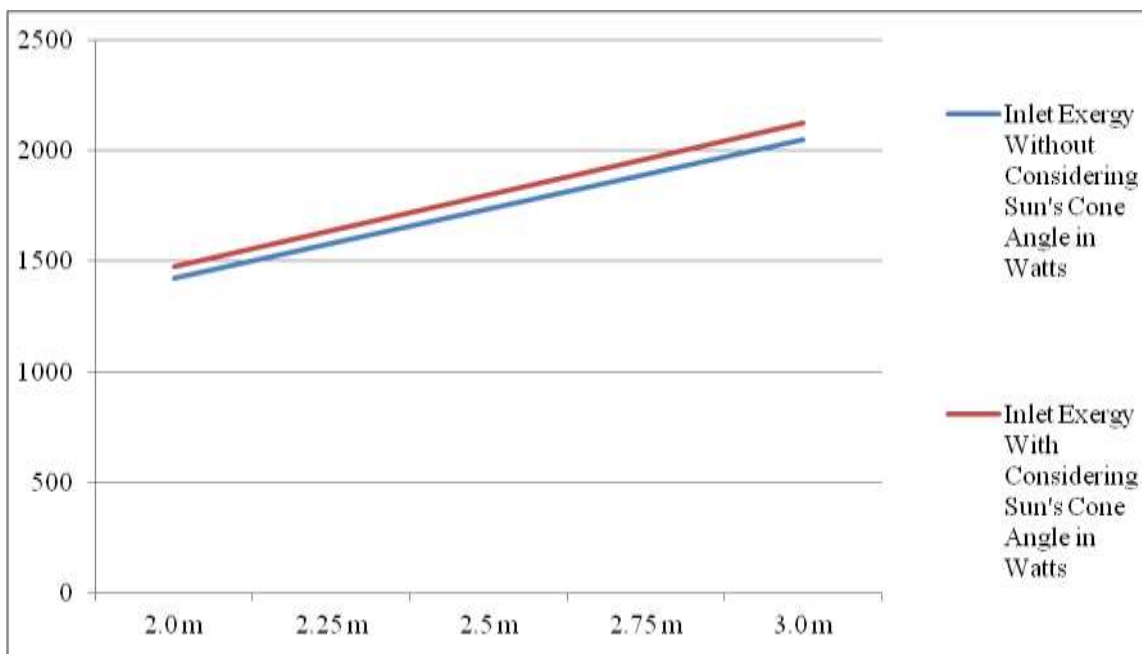
Sr. No.	Inlet Parameter					Outlet Parameter				
	<b>m<sub>f</sub></b> (kg/sec)	<b>T<sub>fi</sub></b> (°C)	<b>T<sub>c</sub></b> (°C)	<b>T<sub>fo</sub></b> (°C)	<b>T<sub>p</sub></b> (°C)	<b>Q<sub>u</sub></b> (W)	<b>D<sub>ib</sub></b> (%)	<b>h<sub>f</sub></b> (W/m <sup>2</sup> K)	<b>S</b> (W/m <sup>2</sup> )	<b>F'</b>
1	<b>0.0019</b>	<b>32</b>	<b>64</b>	57	63.97	75.478	4.5	49.07	19.78	0.95
2	<b>0.0049</b>	<b>32</b>	<b>75</b>	50	74.94	140.15	8.35	25.508	19.78	0.72
3	<b>0.0105</b>	<b>32</b>	<b>55</b>	36	54.973	66.738	3.98	15.981	19.78	0.9
4	<b>0.0142</b>	<b>32</b>	<b>48</b>	35	47.973	67.691	4.03	23.692	19.78	0.87

**Table 16** – At 03:30PM with Mild Steel Engine Oil combination ( $I_b R_b$  472.8  $W/m^2$ )

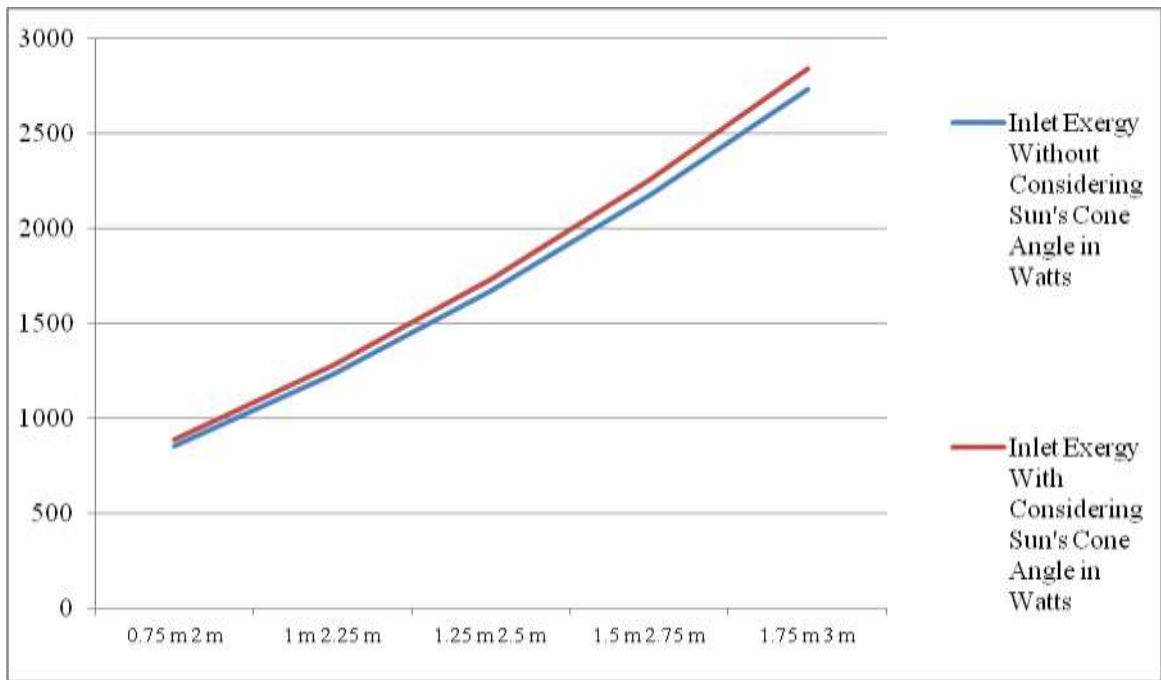
Sr. No.	Inlet Parameter					Outlet Parameter				
	$m_f$ (kg/sec)	$T_{fi}$ (°C)	$T_c$ (°C)	$T_{fo}$ (°C)	$T_p$ (°C)	$Q_u$ (W)	$D_{ib}$ (%)	$h_f$ ( $W/m^2K$ )	$S$ ( $W/m^2$ )	$F'$
1	0.0056	42	56	49	55.97	62.289	4.22	40.49	17.409	0.95
2	0.0032	41	47	45	46.998	20.339	1.38	46.267	17.409	0.74
3	0.015	32	55	36	54.974	95.34	6.45	22.832	17.409	0.67
4	0.0083	38	45	44	44.98	79.132	5.36	360.74	17.409	0.96



**Figure 3** – Inlet exergy curves with different aperture widths for copper-oil arrangement



**Figure 4** – Inlet exergy curves with different pipe lengths for mild steel-water arrangement



**Figure 5** – Inlet exergy curves with different aperture widths and pipe lengths for mild steel-oil arrangement



**Figure 6** – Inlet exergy curves with different mass flow rates for copper-water arrangement



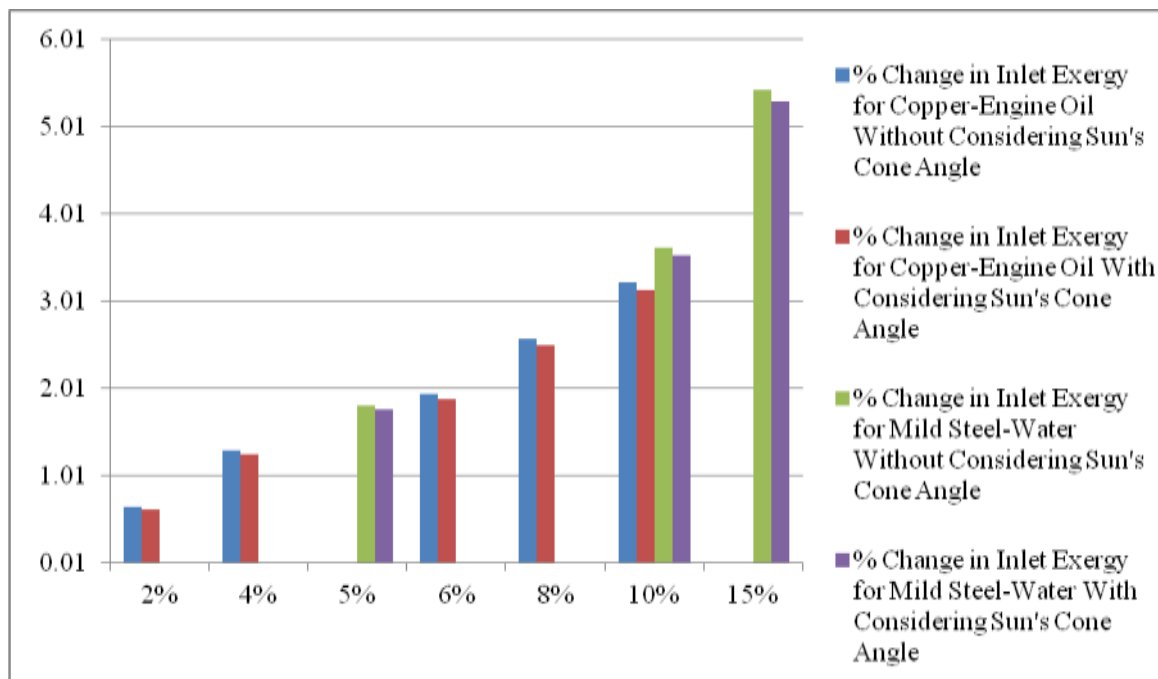


Figure 7 – Inlet exergy v/s change in flow rates

### Conclusion –

Cylindrical parabolic solar collector is fabricated and then experimental analyses are done with various combinations/arrangements. Various arrangements are – (1) copper pipe and engine oil as flowing fluid, (2) copper pipe and water as flowing fluid, (3) mild steel pipe and engine oil as flowing fluid and (4) mild steel pipe and water as flowing fluid. All experimental results are taken at 12:30PM, at 01:30PM, at 02:30PM and at 03:30PM. Experimental results are compared and then concluded as – maximum temperature difference (59°C) of flowing fluid is achieved at 12:30PM with mild steel-engine oil arrangement when flow rate is 0.0024kg/sec but minimum temperature differences (3°C) are found at 02:30PM and at 03:30PM with mild steel-engine oil and copper-engine oil arrangements when flow rates are 0.0142kg/sec and 0.0073kg/sec respectively. When engine oil flows through mild steel pipe with rate of 0.0046kg/sec at 01:30PM then near about one collector efficiency factor is found. Minimum inlet exergy (with and without Sun's cone angle) is found at minimum dimensions of experimental setup (0.75 m aperture width and 2.0 m pipe length) but maximum inlet exergy (with and without Sun's cone angle) is achieved with bigger dimensions of collector (example 1.75m pipe length and 3.0m aperture width). When flow rate of engine oil through copper pipe increases with 2%, 4%, 6%, 8% and 10% then inlet exergy (without considering Sun's cone angle) increases with 0.65%, 1.29%, 1.94%, 2.58% and 3.22% respectively. And when flow rate of engine oil through copper pipe increases with 2%, 4%, 6%, 8% and 10% then inlet exergy (with considering Sun's cone angle) also increases with 0.62%, 1.25%, 1.88%, 2.51% and 3.14% respectively. When flow rate of water through mild steel pipe increases with 5%, 10% and 15% then inlet exergy (without considering Sun's cone angle) increases with 1.81%, 3.63% and 5.44% respectively. And when flow rate of water through mild steel increases with 5%, 10% and 15% then

inlet exergy (with considering Sun's cone angle) also increases with 1.76%, 3.53% and 5.30% respectively. Maximum instantaneous efficiency of the collector is found at 12:30PM with copper-water arrangement when flow rate is 0.0166kg/sec. It means this arrangement is best arrangement.

### List of Symbols, Abbreviations, Nomenclatures –

- $A_s$  = Surface area of pipe ( $m^2$ )
- $C_p$  = Specific heat for flowing fluid ( $J/kg-K$ )
- $F'$  = Collector efficiency factor
- $h_f$  = Convective heat transfer coefficient ( $W/m^2K$ )
- $I_b R_b$  = Instantaneous/hourly beam radiation on a surface ( $W/m^2$ )
- $K$  = Thermal conductivity of the material ( $W/mK$ )
- $L_c$  = Characteristic length of pipe (metre)
- $m_f$  - Mass flow rate of flowing fluid ( $kg/s$ )
- $Qu$  = Heat transfer rate ( $W$ )
- $S$  = Absorbed flux ( $W/m^2$ )
- $T_{fo}, T_{fi}$  = Outlet and inlet fluid temperatures ( $K$ )
- $T_c, T_p$  = Outlet and inlet surface temperatures of pipe material ( $K$ )
- $T_a$  = Ambient temperature ( $K$ )
- $U_l$  = Overall heat loss coefficient ( $W/m^2K$ )
- $W$  = Aperture width ( $m$ )
- $\eta_{ib}$  = Instantaneous efficiency (%)
- $Y$  = Intercept factor
- $\alpha$  = Absorbitivity of the pipe material

$\tau$  = *Transmissivity of the pipe material*

$\rho$  = *Specular reflectivity on the concentrator surface*

EMSES), Energy Procedia, Volume 56, pp 237 – 246, 2014.

#### Reference –

- [1] Sukhatme S. P., Nayak J. K., “Solar Energy Principles of Thermal Collection and Storage”, 3<sup>rd</sup> Edition, The McGraw Hill Companies, New Delhi, India, 2011.
- [2] Mustapha D., Abdallah L., Houria H. B., “Analysis of the Energetic Feasibility of Parabolic Trough Collectors Integrated in Solar Towers in Adrar Area”, Energy Procedia, Volume 36, pp 1085 – 1100, 2013.
- [3] Kumar K. V. P., Srinath T., Reddy V., “Design, Fabrication and Experimental Testing of Solar Parabolic Trough Collectors with Automated Tracking Mechanism”, International Journal of Research in Aeronautical and Mechanical Engineering, Volume 01, Issue 04, pp 37-55, 2013.
- [4] Tzivaidis C., Bellos B., Korres D., Antonopoulos K. A., Mitsopoulos G., “Thermal and Optical Efficiency Investigation of A Parabolic Trough Collector”, Case Studies in Thermal Engineering, Volume 06, pp 226-237, 2015.
- [5] Wang F., Feng H., Zhao J., Li W., Zhang F., Liu R., “Performance Assessment of Solar Assisted Absorption Heat Pump System with Parabolic Trough Collectors”, Energy Procedia, Volume 70, pp 529 – 536, 2015.
- [6] Almasabi A., Alobaidli A., Zhang T. J., “Transient Characterization of Multiple Parabolic Trough Collector Loops in A 100 MW CSP Plant for Solar Energy Harvesting”, Energy Procedia, Volume 69, pp 24 – 33, 2015.
- [7] Liu X., Huang J., Mao Q., “Sensitive Analysis for the Efficiency of a Parabolic Trough Solar Collector Based on Orthogonal Experiment”, International Journal of Photoenergy, Volume 2015, Article ID 151874, pp 1-7, <http://dx.doi.org/10.1155/2015/151874>.
- [8] Mohamad A., Orfi J., Alansary H., “Heat Losses from Parabolic Trough Solar Collectors”, International Journal of Energy Research, Volume 38, pp 20-28, 2014.
- [9] Petela R., Exergy of Undiluted Thermal Radiation, Solar Energy, Volume 76, Issue 06, pp 469-488, 2003.
- [10] Zarza E., “Medium Temperature Solar Concentrators (Parabolic-Throughs Collectors)”, Solar Energy Conversion and Photoenergy Systems, Volume 01, pp 01-09.
- [11] Padilla R. V., Fontalvo A., Demirkaya G., Martinez A., Quiroga A. G., “Exergy analysis of parabolic trough solar receiver”, Applied Thermal Engineering, Volume 67, pp 1-8, 2014.
- [12] Chamsa-ard W., Sukchai S., Sonsaree S., Sirisamphanwong C., “Thermal Performance Testing of Heat Pipe Evacuated Tube with Compound Parabolic Concentrating Solar Collector by ISO 9806 - 1”, 11th Eco-Energy and Materials Science and Engineering (11th